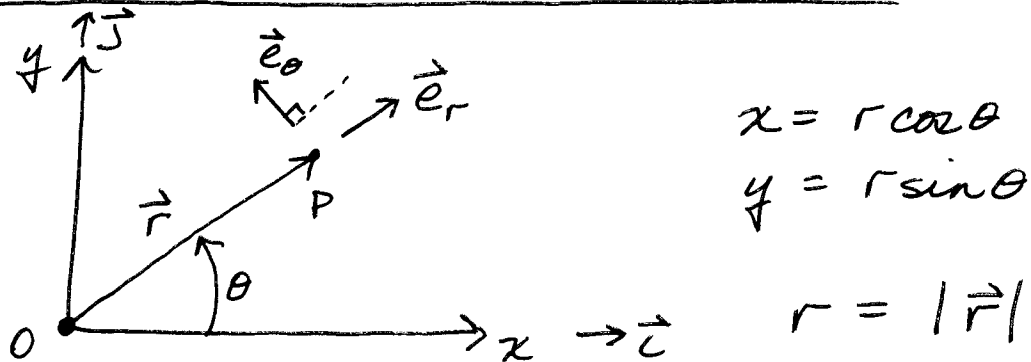


Kinematics in Polar Coordinates



$$\begin{aligned}\vec{i} &= \vec{e}_r \cos \theta - \vec{e}_\theta \sin \theta \\ \vec{j} &= \vec{e}_r \sin \theta + \vec{e}_\theta \cos \theta\end{aligned}$$

Similarly

$$\begin{aligned}\vec{e}_r &= \vec{i} \cos \theta + \vec{j} \sin \theta \\ \vec{e}_\theta &= -\vec{i} \sin \theta + \vec{j} \cos \theta\end{aligned}$$

Recall

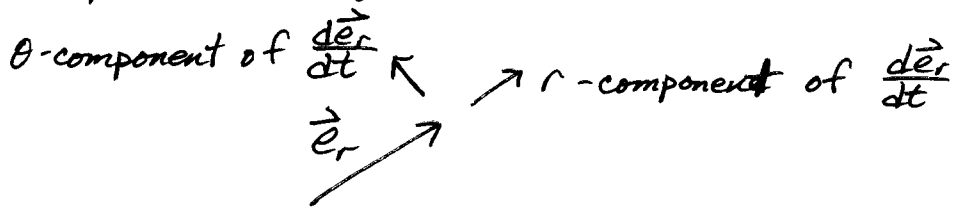
$$\begin{aligned}\vec{r} &= x \vec{i} + y \vec{j} \\ &= (r \cos \theta)(\vec{e}_r \cos \theta - \vec{e}_\theta \sin \theta) \\ &\quad + (r \sin \theta)(\vec{e}_r \sin \theta + \vec{e}_\theta \cos \theta) \\ &= r \cos^2 \theta \vec{e}_r - r \cos \theta \sin \theta \vec{e}_\theta \\ &\quad + r \sin^2 \theta \vec{e}_r + r \sin \theta \cos \theta \vec{e}_\theta \\ &= r (\cos^2 \theta + \sin^2 \theta) \vec{e}_r = r \vec{e}_r\end{aligned}$$

Note: \vec{e}_r is the unit vector in the r direction so as we should expect

$$\boxed{\vec{r} = r \vec{e}_r}$$

Now: $\vec{v} = \frac{d\vec{r}}{dt} = \frac{d}{dt} (r \vec{e}_r)$
 $= \underbrace{\frac{dr}{dt}}_r \vec{e}_r + r \underbrace{\frac{d\vec{e}_r}{dt}}_?$

Recall: Any vector, including $\frac{d\vec{e}_r}{dt}$, can be decomposed into orthogonal components. So we can take components of $\frac{d\vec{e}_r}{dt}$ in the \vec{e}_r and \vec{e}_θ directions.



But \vec{e}_r is a unit vector so its length cannot change. Therefore the r -component of $\frac{d\vec{e}_r}{dt}$ must be zero. In general the rate of change of any unit vector will always be perpendicular to itself (or zero as in the case of $\vec{i}, \vec{j}, \vec{k}$).

$$\begin{aligned} \frac{d}{dt}(\vec{e}_r) &= \frac{d}{dt}(\vec{i} \cos\theta + \vec{j} \sin\theta) \\ &= \frac{d\vec{i}}{dt} \cos\theta + \vec{i} \frac{d}{dt}(\cos\theta) + \frac{d\vec{j}}{dt} \sin\theta + \vec{j} \frac{d}{dt}(\sin\theta) \\ &= \vec{i} \underbrace{(-\sin\theta)}_{\frac{d}{d\theta}(\cos\theta)} \frac{d\theta}{dt} + \vec{j} \underbrace{(\cos\theta)}_{\frac{d}{d\theta}(\sin\theta)} \frac{d\theta}{dt} \end{aligned}$$

$$\therefore \frac{d\vec{e}_r}{dt} = \dot{\theta} \underbrace{(-\vec{i} \sin\theta + \vec{j} \cos\theta)}_{\vec{e}_\theta}$$

$$\boxed{\frac{d\vec{e}_r}{dt} = \dot{\theta} \vec{e}_\theta}$$

$$\begin{aligned} \text{Similarly: } \frac{d\vec{e}_\theta}{dt} &= \frac{d}{dt} (-\vec{i} \sin\theta + \vec{j} \cos\theta) \\ &= -\vec{i} (\cos\theta) \dot{\theta} + \vec{j} (-\sin\theta) \dot{\theta} \\ &= -\dot{\theta} \underbrace{(\vec{i} \cos\theta + \vec{j} \sin\theta)}_{\vec{e}_r} \end{aligned}$$

$$\boxed{\frac{d\vec{e}_\theta}{dt} = -\dot{\theta} \vec{e}_r}$$

Back to \vec{v} .

$$\therefore \boxed{\vec{v} = \dot{r} \vec{e}_r + r \dot{\theta} \vec{e}_\theta}$$

\dot{r} is the normal or radial component of \vec{v} and $r \dot{\theta}$ is the tangential or θ component of \vec{v} , i.e.

$$v_n = v_r = \dot{r} \quad \text{and}$$

$$v_t = v_\theta = r \dot{\theta}$$

~~Now~~ Now for \vec{a} .

$$\begin{aligned}\vec{a} &= \frac{d\vec{v}}{dt} = \frac{d}{dt} (\dot{r} \vec{e}_r + r \dot{\theta} \vec{e}_\theta) \\ &= \underbrace{\frac{d\dot{r}}{dt}}_{\ddot{r}} \vec{e}_r + \dot{r} \underbrace{\frac{d\vec{e}_r}{dt}}_{\dot{\theta} \vec{e}_\theta} + \underbrace{\frac{dr}{dt}}_{\dot{r}} \dot{\theta} \vec{e}_\theta + r \underbrace{\frac{d\dot{\theta}}{dt}}_{\ddot{\theta}} \vec{e}_\theta + r \dot{\theta} \underbrace{\frac{d\vec{e}_\theta}{dt}}_{-\dot{\theta} \vec{e}_r}\end{aligned}$$

$$\vec{a} = (\ddot{r} - r\dot{\theta}^2) \vec{e}_r + (r\ddot{\theta} + 2\dot{r}\dot{\theta}) \vec{e}_\theta$$

radial / normal acceleration $a_r = a_n = \ddot{r} - r\dot{\theta}^2$

$\ddot{r} \rightarrow$ accelerating along a ray

$r\dot{\theta}^2 = \frac{(r\dot{\theta})^2}{r} = \frac{v_t^2}{r} \rightarrow$ centripetal acceleration

tangential / θ acceleration $a_t = a_\theta = r\ddot{\theta} + 2\dot{r}\dot{\theta}$

$r\ddot{\theta} \rightarrow$ accelerating around a circle

$2\dot{r}\dot{\theta} \rightarrow$ unintuitive "Coriolis acceleration"

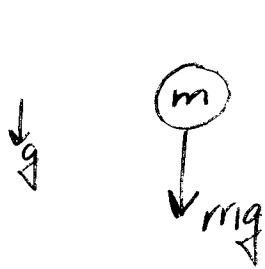
Kinetics

$\Sigma \vec{F} = m\vec{a} \rightarrow$ leads to equations of motion that are in many cases described by differential equations.

Projectiles

Projectiles act only under the force of gravity (if we neglect wind resistance).

FBD



$$\Sigma F_x = 0 = ma_x$$

$$\Sigma F_y = -mg = ma_y$$

$$x \rightarrow a_x = 0 \rightarrow v_x = C_1 \rightarrow x = C_1 t + C_2$$

Apply initial conditions : $x(t=0) = x_0$
 $v_x(t=0) = \dot{x}(t=0) = v_{x0}$

$$\therefore C_1 = v_{x0} \quad \text{and} \quad C_2 = x_0$$

$$x = x_0 + v_{x0} t, \quad \text{i.e. constant velocity motion in } x\text{-direction}$$

$$y \rightarrow a_y = -g \rightarrow v_y = -gt + C_3 \rightarrow y = -\frac{1}{2}gt^2 + C_3t + C_4$$

Apply initial conditions: $y(t=0) = y_0$

$$v_y(t=0) = \dot{y}(t=0) = v_{y0}$$

$$\therefore C_3 = v_{y0} \text{ and } C_4 = y_0$$

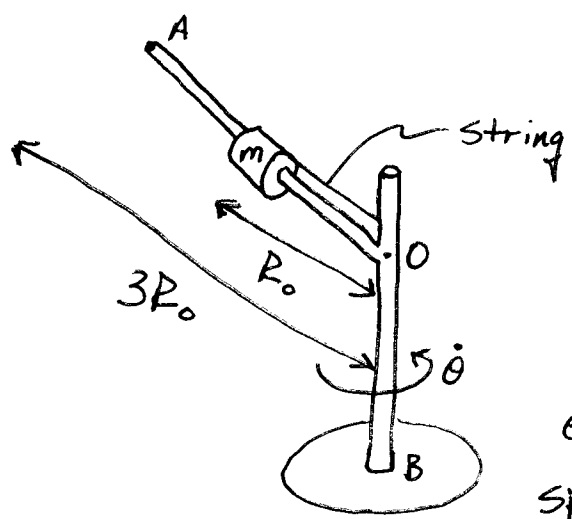
$$\therefore y = y_0 + v_{y0}t - \frac{1}{2}gt^2$$

Hence the position as a function of time for a projectile can be given as

$$\begin{aligned} x &= x_0 + v_{x0}t \\ y &= y_0 + v_{y0}t - \frac{1}{2}gt^2 \end{aligned}$$

Projectile motion with gravity in -y direction.

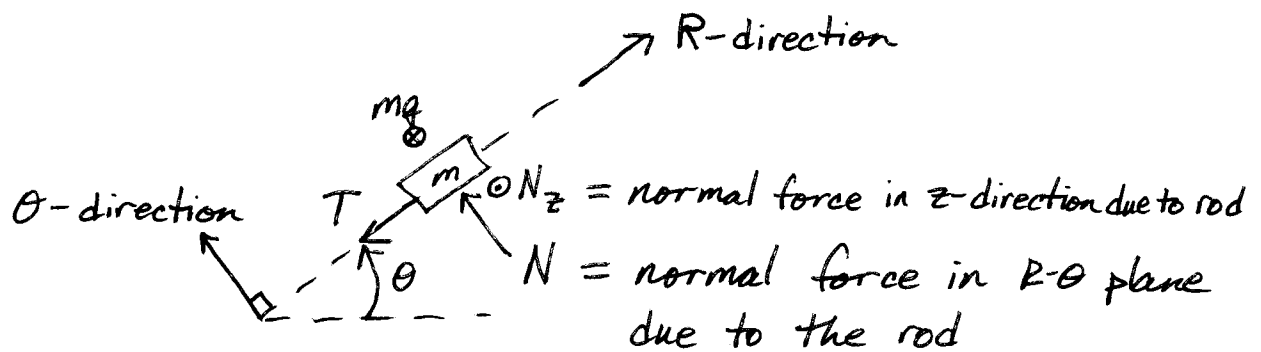
Example Problem 13.59



$\dot{\theta} = \omega = \text{constant}$
 frictionless slider of mass m . String cut at time $t=0$.
 Determine T in string before cut, \ddot{R} after cut, equation of motion for $R(t)$, speed when slider reaches A.

We will be using $\Sigma \vec{F} = m\vec{a}$. Any time you see $\Sigma \vec{F}$ you should automatically start drawing a FBD.

Looking down on the rod OB



$$\Sigma F_z = ma_z \rightarrow N_z - mg = m\vec{a}_z^0 \therefore N_z = mg$$

$$\Sigma F_\theta = ma_\theta \rightarrow N = m(R\ddot{\theta} + z\dot{R}\dot{\theta})$$

$$\therefore N = 2m\dot{R}\omega \quad \dot{R} = ?$$

$$\Sigma F_R = ma_R \rightarrow -T = m(\ddot{R} - R\dot{\theta}^2)$$

Just prior to cutting the string $R = R_0$
 $\dot{R} = 0$
 $\ddot{R} = 0$

$$\therefore N = 2m\dot{R}\omega = 0 \text{ and } -T = m(0 - R_0\omega^2)$$

$$\therefore T = mR_0\omega^2 \text{ before cut}$$

Just after the cut $T = 0$
 $R = R_0$
 $\dot{R} = 0$

Note that when we do not have impacts, velocity components cannot change instantaneously. However, acceleration components can.

$$\begin{aligned} \therefore N &= Zm \cdot 0 \cdot \omega = 0 \\ 0 &= m(\ddot{R} - R_0 \omega^2) \rightarrow \ddot{R} = R_0 \omega^2 \end{aligned} \left. \vphantom{\begin{aligned} \therefore N &= Zm \cdot 0 \cdot \omega = 0 \\ 0 &= m(\ddot{R} - R_0 \omega^2) \rightarrow \ddot{R} = R_0 \omega^2 \end{aligned}} \right\} \begin{array}{l} \text{Just after} \\ \text{cut.} \end{array}$$

The equation of motion after the string has been cut is from $\Sigma F_R = ma_R$, i.e.

$$-T = 0 = m(\ddot{R} - R\dot{\theta}^2)$$

$$\therefore \overset{\substack{\uparrow \\ \text{after} \\ \text{cut}}}{\ddot{R}} - \omega^2 R = 0 \text{ or } \boxed{\frac{d^2 R}{dt^2} - \omega^2 R = 0}$$

We were asked for the speed at A.

$$\text{Recall } \vec{v} = \dot{R} \vec{e}_r + R \dot{\theta} \vec{e}_\theta$$

$$\text{at A } R = 3R_0, \dot{\theta} = \omega, \dot{R} = ?$$

We will go through 2 methods to get \dot{R} .

Method 1 : Solve the ODE

$$\frac{d^2 R}{dt^2} - \omega^2 R = 0 \rightarrow \text{2nd order, linear ODE} \\ \text{with constant coefficients}$$

Assume a solution of the form $R(t) = A \exp[\alpha t]$

$$\frac{dR}{dt} = A \alpha \exp[\alpha t]$$

$$\frac{d^2 R}{dt^2} = A \alpha^2 \exp[\alpha t]$$

Plug into our equation: $A \alpha^2 \exp[\alpha t] - A \omega^2 \exp[\alpha t] = 0$

$$\therefore -\omega^2 + \alpha^2 = 0 \\ \alpha = \pm \omega$$

Then the general solution is $R(t) = A \exp[\omega t] + B \exp[-\omega t]$

$$\dot{R}(t) = A \omega \exp[\omega t] - B \omega \exp[-\omega t]$$

Apply initial conditions to determine A & B .

$$R(t=0) = R_0 \quad \text{and} \quad \dot{R}(t=0) = 0$$

$$\therefore A + B = R_0 \quad \omega(A - B) = 0$$

$$\hookrightarrow A = B = \frac{R_0}{2} \quad \longleftarrow$$

So $R(t) = \frac{R_0}{2} [\exp(\omega t) + \exp(-\omega t)] = R_0 \cosh(\omega t)$

$$\text{at } A, R = 3R_0 \rightarrow R(t_A) = R_0 \cosh(\omega t_A) = 3R_0$$

$$\therefore \omega t_A = \operatorname{arccosh}(3) = 1.763$$

$$\therefore \dot{R}(t_A) = R_0 \omega \sinh(\omega t_A) = 2.828 R_0 \omega$$

$$\therefore \vec{V}_A = 2.828 R_0 \omega \vec{e}_r + 3 R_0 \omega \vec{e}_\theta$$

$$V_A = R_0 \omega \sqrt{2.828^2 + 3^2} = 4.123 R_0 \omega$$

Method 2 : Integration Tricks

Recall $\ddot{R} = \omega^2 R$ from our equation of motion

$$\text{but } \ddot{R} = \frac{d\dot{R}}{dt} = \frac{d\dot{R}}{dR} \frac{dR}{dt} = \dot{R} \frac{d\dot{R}}{dR}$$

Combine these two equations to get

$$\dot{R} \frac{d\dot{R}}{dR} = \omega^2 R \quad (\text{Similar to } v \frac{dv}{ds} = f(s))$$

$$\text{Integrate } \rightarrow \int_0^{\dot{R}_A} \dot{R} d\dot{R} = \int_{R_0}^{3R_0=R_A} \omega^2 R dR$$

\uparrow Initial condition

$$\therefore \left. \frac{1}{2} \dot{R}^2 \right|_0^{\dot{R}_A} = \left. \frac{1}{2} \omega^2 R^2 \right|_{R_0}^{3R_0}$$

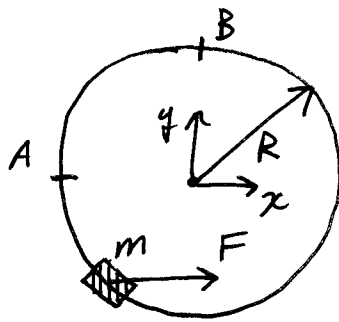
$$\frac{1}{2} \dot{R}_A^2 = \frac{1}{2} \omega^2 (9R_0^2 - R_0^2)$$

$$\therefore \dot{R}_A = \sqrt{8} \omega R_0 = 2.828 \omega R_0$$

$$\begin{aligned} \text{then } \vec{v}_A &= \dot{R}_A \vec{e}_r + R_A \omega \vec{e}_\theta \\ &= 2.828 \omega R_0 \vec{e}_r + 3 R_0 \omega \vec{e}_\theta \end{aligned}$$

$$\therefore v_A = \sqrt{8^2 + 3^2} \omega R_0 = \sqrt{17} \omega R_0 = 4.123 \omega R_0$$

Example Problem 13.55

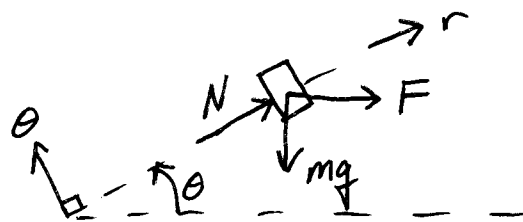


m starts from rest at A.
What is the smallest F that
can cause the slider to
reach B?

F always acts in positive x.

$$\Sigma \vec{F} = m \vec{a} \rightarrow \text{Draw FBD}$$

Let's draw our FBD at a positive θ angle.



(111)

$$\Sigma F_r = ma_r \rightarrow N + F \cos \theta - mg \sin \theta = m(\ddot{r} - R\dot{\theta}^2)$$

$$\Sigma F_\theta = ma_\theta \rightarrow -F \sin \theta - mg \cos \theta = m(R\ddot{\theta} + 2\dot{r}\dot{\theta})$$

but $R = \text{constant} \rightarrow \dot{r} = 0$ and $\ddot{r} = 0$

$$\therefore N + F \cos \theta - mg \sin \theta = -mR\dot{\theta}^2 \quad \text{①}$$

and

$$-F \sin \theta - mg \cos \theta = mR\ddot{\theta} \quad \text{②}$$

We want to find an equation that looks like $\ddot{\theta} = f(\theta)$ because if we do then we can use our integration trick that tells us

$$\ddot{\theta} = \frac{d\dot{\theta}}{dt} = \frac{d\dot{\theta}}{d\theta} \frac{d\theta}{dt} = \dot{\theta} \frac{d\dot{\theta}}{d\theta} = f(\theta)$$

$$\text{then } \int_{\dot{\theta}_i}^{\dot{\theta}_f} \dot{\theta} d\dot{\theta} = \int_{\theta_i}^{\theta_f} f(\theta) d\theta$$

Well, the θ equation looks like this, i.e.

$$\ddot{\theta} = \dot{\theta} \frac{d\dot{\theta}}{d\theta} = -\frac{F}{mR} \sin \theta - \frac{g}{R} \cos \theta$$

\therefore with $\dot{\theta}_i = \dot{\theta}_A = 0$ (starts from rest)

$$\theta_A = -\pi$$

$\dot{\theta}_f = \dot{\theta}_B = 0$ (just reaches B)

$$\theta_B = \frac{\pi}{2}$$

We then can set up our integral as

$$\int_0^0 \dot{\theta} d\dot{\theta} = \int_{-\pi}^{\pi/2} -\frac{F}{mR} \sin\theta - \frac{g}{R} \cos\theta d\theta$$

$$0 = \left[+\frac{F}{mR} \cos\theta - \frac{g}{R} \sin\theta \right]_{-\pi}^{\pi/2}$$

$$= \frac{F}{mR} (0 - -1) - \frac{g}{R} (1 - 0)$$

$$\therefore +\frac{F}{mR} - \frac{g}{R} = 0$$

$$F = mg$$

Hence, the smallest constant horizontal force F that will allow the slider to reach B is $F = mg$.